

# VANED VS. VANELESS DIFFUSOR IN COMPRESSOR STAGES

By Ryan Montero, Lead Aerodynamicist

When designing a centrifugal compressor stage the choice in diffuser design must be considered in clean sheet designs and rerate cases. In general, these diffusers fall into two categories: vaned and vaneless.

The job of the diffuser section of a stage is to convert the dynamic pressure (velocity head) coming out of the impeller into static pressure. The velocity will be reduced going into the crossover bend, or volute depending on the stage, reducing the amount of loss taken going into the final sections of the stage. The diffusion done in a vaneless space is primarily done through area increase as the flow goes radially outward. For a given diffuser width, as the radius increases so will the area. A vaned diffuser can achieve diffusion in two ways; through the increase in area as radius increases and through use of diffusion over the vane. The vane acts as a diffusing agent itself through the pressure field created on the airfoil surface.

Diffusers can also be designed to aid in surge avoidance.

As the flow in a stage decreases, the area increase in a vaneless space becomes more of a hindrance than a benefit; and as the flow angle decreases below a stable value the flow in the diffuser can reverse and surge can occur. Vaneless diffusers can be “pinched”, meaning the diffuser width decreases with radius, in order to attempt to avoid this. The issue with this is of course that the area increase that the diffuser depends on may not exist depending on the severity of the pinch required. Vaned diffusers can tune diffuser angles to turn the flow for increasing angle to avoid surge. In this way the vaned spaces still get the diffusion of the area increase and vane diffusion, all while helping to create head margin to surge. Figure 1 shows a vaned and vaneless diffuser tuned to create similar curve shape and head margin to surge, these diffusers follow identical impellers. In this case the vaneless diffuser is pinched to create the same head margin as the vaned diffuser. Notice how the static pressure is higher at the exit of the vaned diffuser where adequate diffusion has taken place.

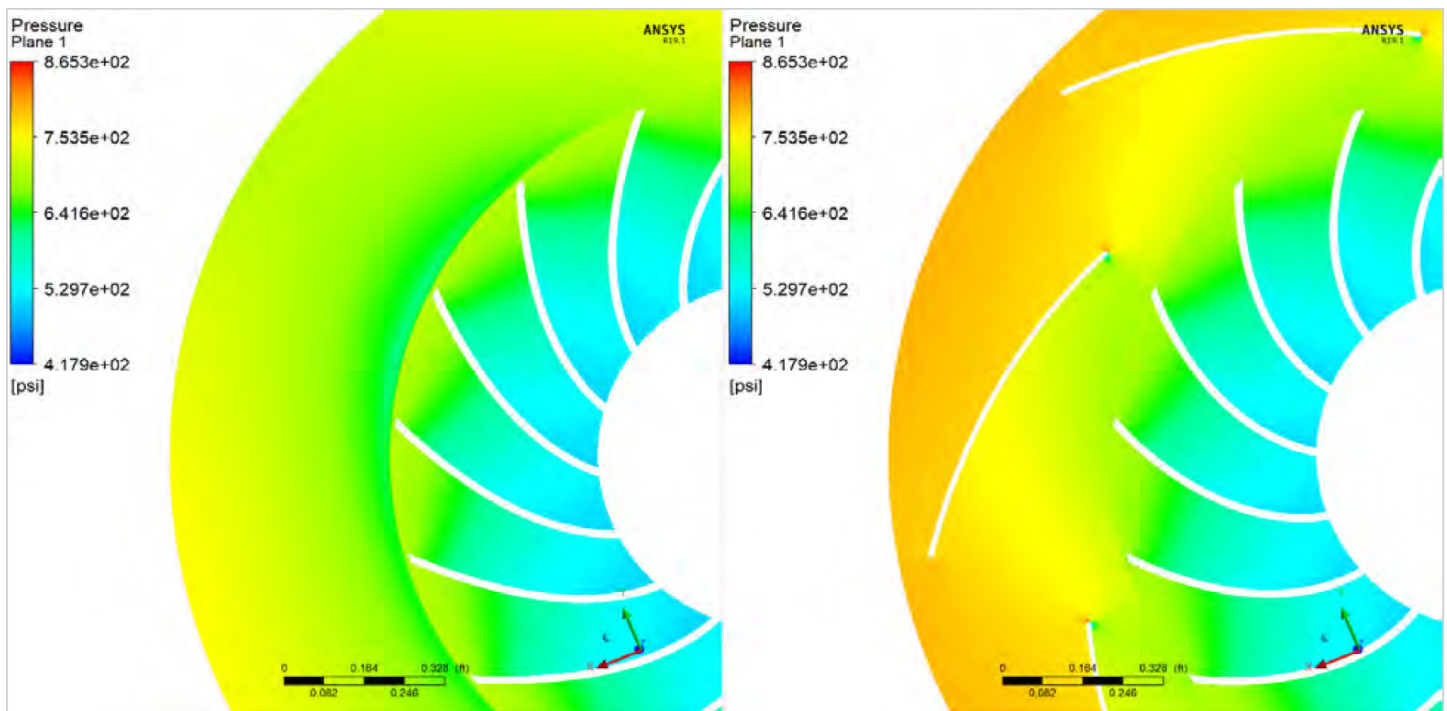


Figure 1: Pressure distributions determined by CFD results for vaned (right) and vaneless (left) diffusers designed for similar impellers/stages

Both vaneless and vaneless diffuser designs have benefits and drawbacks to their implementation.

### Vaneless Diffusor

- Better flow range
- Better performance off design
- Reduces aeroacoustic effects of pressure pulsations from impeller

### Vaned Diffusor

- Better performance near design point
- Lower velocity going into discharge volute
- Higher efficiency

Figure 2 illustrates this by directly comparing head and efficiency curves for similar stages with both vaneless and vaneless diffusers. As anticipated, the vaneless diffuser curve is slightly less efficient near the design point, but the compressor has a greater operating range and has more sustained performance off-design. The vaneless by comparison has shorter range, but is more efficient at the design point. This makes vaneless diffusers a better choice where the user anticipates narrow flow ranges during operation, where the compressor can be kept near the design point and can leverage these efficiency benefits of the vaneless diffuser.

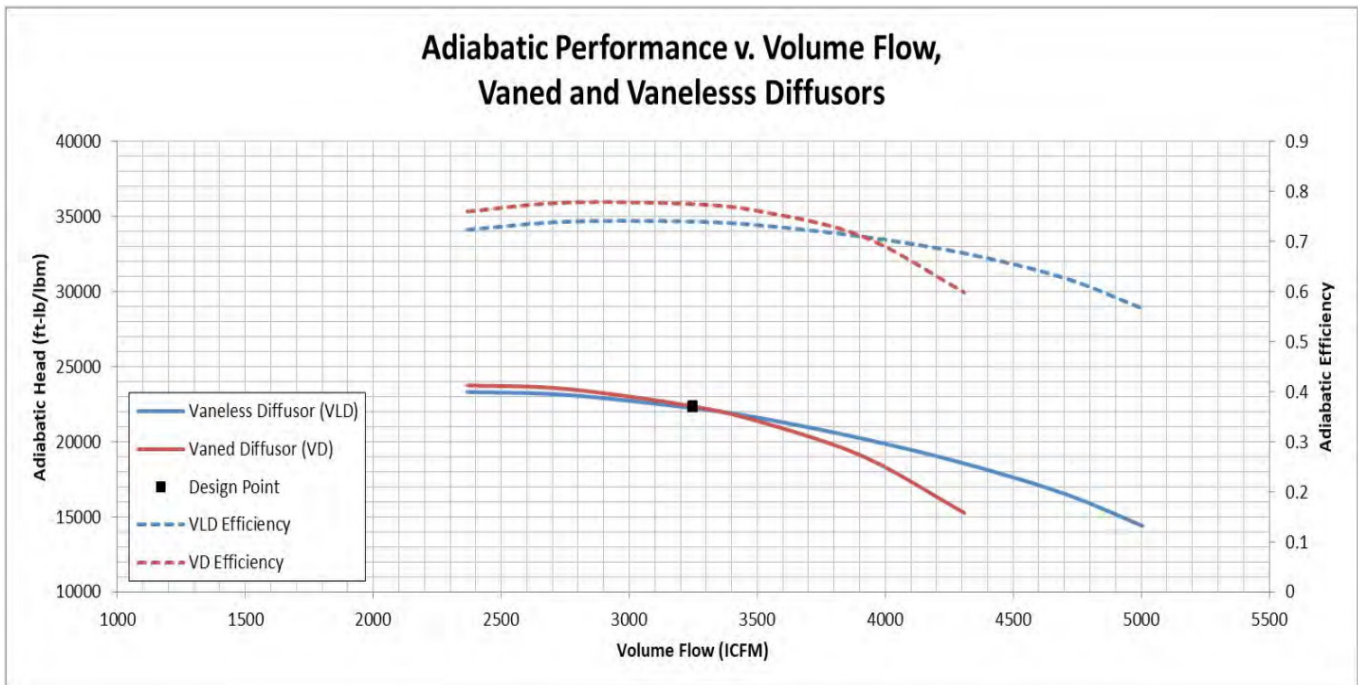


Figure 2: Performance curves for stages designed for the same operating point with vaneless and vaneless diffuser designs

#### For more information:

Ryan Montero, Lead Aerodynamicist  
Email: [rmontero@rotatingmachinery.com](mailto:rmontero@rotatingmachinery.com)  
Tel: 484-821-0702

#### Headquarters

2760 Baglyos Cir.  
Bethlehem, PA 18020

#### Houston Office

16676 Northchase Dr., Ste 400  
Houston, TX 77060



[rotatingmachinery.com](http://rotatingmachinery.com)

Tel: 484-821-0702

Parts: [rms@rotatingmachinery.com](mailto:rms@rotatingmachinery.com)

Rotating Machinery Services, Inc. | 2760 Baglyos Circle, Bethlehem, PA 18020 | Tel: 484-821-0702